# DESIGN, MANUFACTURING, TESTING AND OPTIMIZATION OF BOLT LOADED VARIABLE-AXIAL COMPOSITE LAMINATES





TWENTY-THIRD INTERNATIONAL CONFERENCE ON COMPOSITE MATERIALS (ICCM23)

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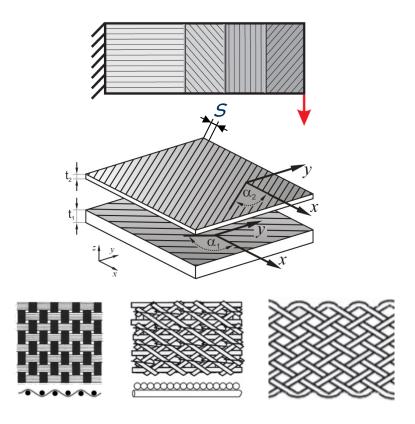
# Fiber-reinforced plastics (FRP)

**ipf** 

Multi-axial vs. variable-axial

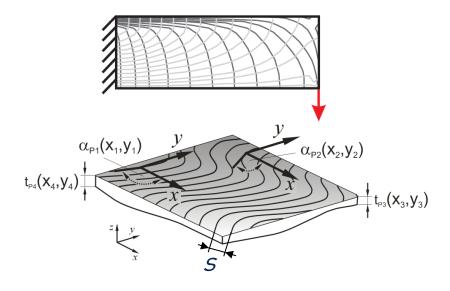
# Composite design 1.0

multi-axial



#### Composite design 2.0

variable-axial



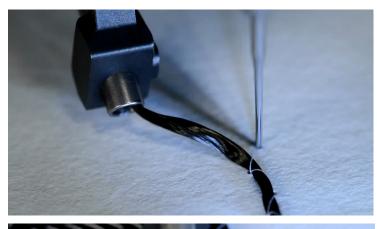
 $\alpha \neq const.$ 

s ≠ const. t = f(s)

# Fiber-reinforced plastics (FRP)



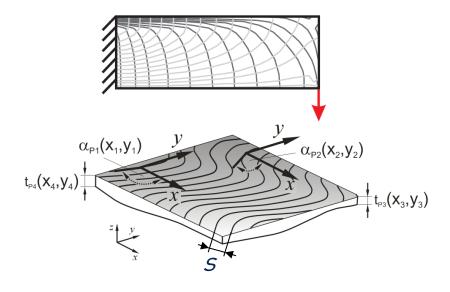
Variable-axial FRP by Tailored Fiber Placement (TFP)





### Composite design 2.0

variable-axial



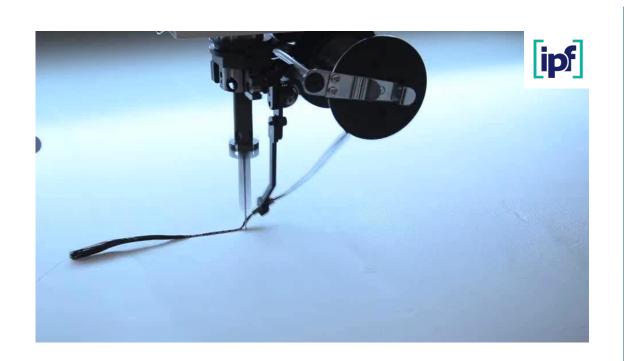
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# Fiber-reinforced plastics (FRP)

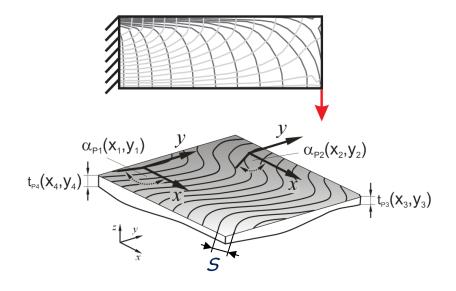


Variable-axial FRP by Tailored Fiber Placement (TFP)



#### Composite design 2.0

variable-axial



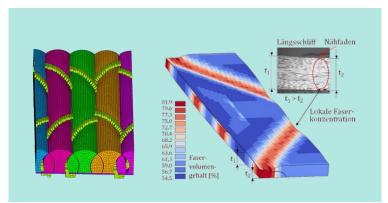
 $\alpha \neq const.$ 

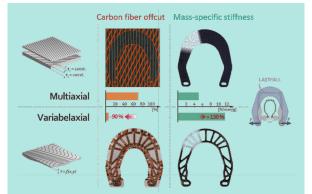
 $s \neq const.$ t = f(s)

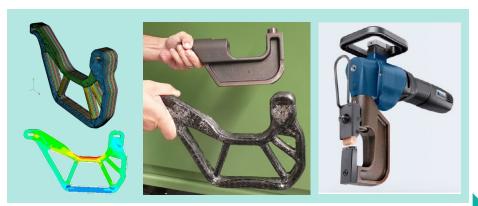
# Variable-axial composites made by Tailored Fiber Placement



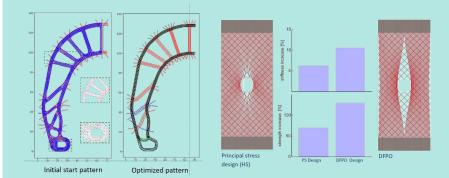
#### From basic research to industrial deployment







#### TRL1 TRL7



Multi-scale modelling and optimization methods



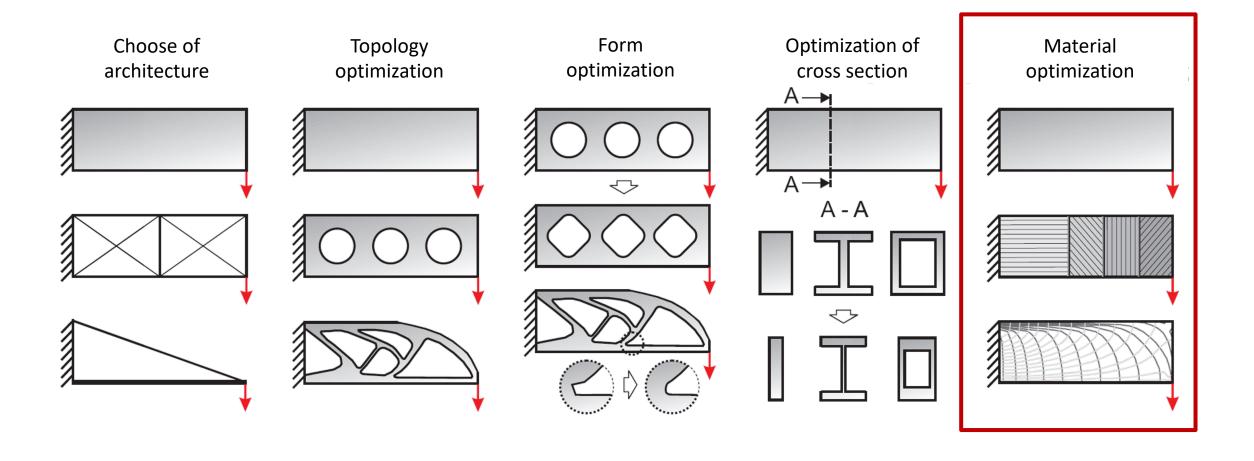
Process chain for variableaxial composites



Development of industrial components

# How to design variable-axial composite structures?



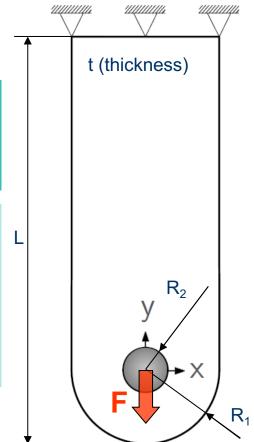


# Analysis and optimization of bolted joints made by variable-axial laminates



What in-plane fiber orientation strategy shows advantageous mechanical performance?

- Calibration of progressive damage simulation for structural optimization by experimental research
- No consideration of out-of-plane reinforcements



#### **Dimensions of specimen**

 $R_1 = 25 \text{ mm}, R_2 = 7.5 \text{ mm},$  $L = 170 \text{ mm}, t \approx 5 \text{ mm}$ 

# **Bolted joints with tailored fiber orientation in literature**



### 1997 Composites Part A 28A (1997) 619–625 © 1997 Published by Elsevier Science Limited Printed in Great Britain. All rights reserved 1359-835X/97/\$17.00 Tailored fibre placement to minimise stress concentrations P. J. Crothers\*\*, K. Drechsler\*, D. Feltin\*, I. Herszberg\* and T. Kruckenberg\* \*The Sir Lawrence Wackett Centre for Aerospace Design Technology, Royal Melbourn Institute of Technology, P.O. Box 2476V, Melbourne, Vic. 300. Australia \*Daimler Benz AG, Research and Technology, P.O.Box 80 04 65, 81663 Munich, Germany °Institute for Polymer Research Dresden (IPF-Dresden), Hohe Straße 6, D-01069 Institute on conjune nesental pressure in representations. The Composite Structures (CRC-ACS) Ltd, 506 Lorimer St., Fishermens Bend, Vic. 3207, Australia (Received 16 February 1996; accepted 8 November 1996) Tension Plate Tension Plate: Single-bolt loading. Open hole. Single-bolt loading Bolt as rigid body Bolt as non-rigid body Pattern from nast study

Tension Strut:

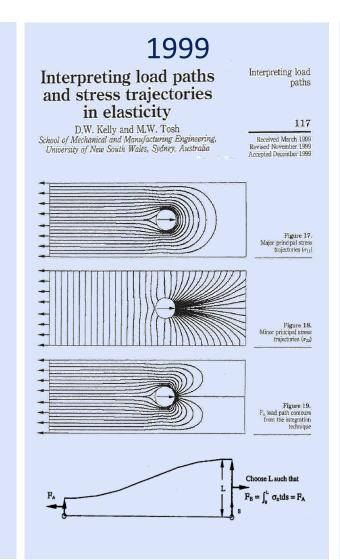
Double-bolt loading

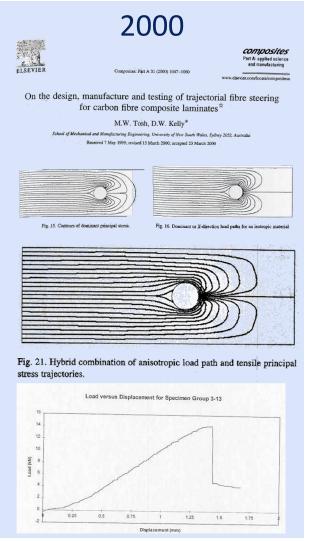
Bolt as rigid body.

Tension Plate &

Tension Strut:

Single and Double-bolt Loading





#### 2002



Composite Structures 57 (2002) 377-383

COMPOSITE STRUCTURES

www.elsevier.com/locate/compstruc

Strength improvement by fibre steering around a pin loaded hole

R. Li a, D. Kelly a.\*, A. Crosky b

School of Mechanical and Manufacturing Engineering, University of New South Wales, Sydney NSW 2052, Australia b School of Materials Science and Engineering, University of New South Wales, Sydney NSW 2052, Australia

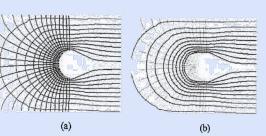


Fig. 9. Reduction of fibre over stacking: (a) FEA pattern; (b) modified pattern.

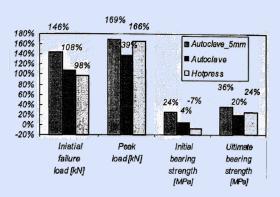


Fig. 10. Effect of fibre steering on strength of the carbon fibre bolted joints.

Tension Strut:

Double-bolt loading

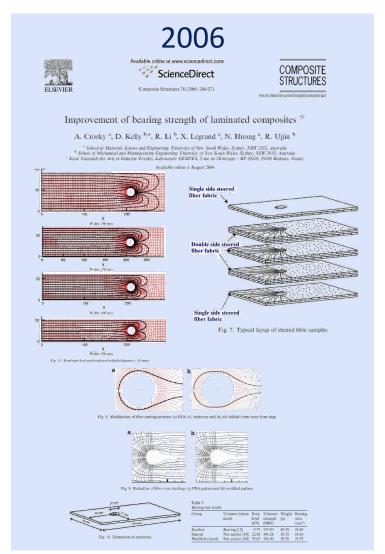
Bolt as non-rigid body.

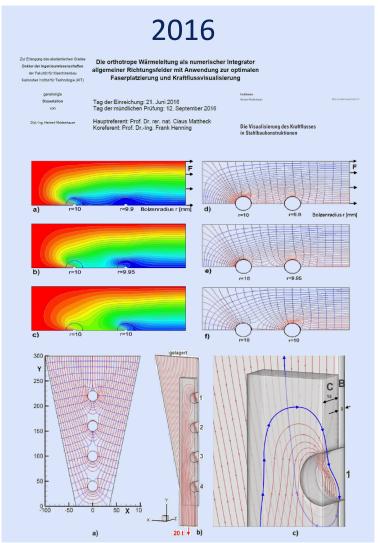
Figure 5 Patterns for TFP reinforcement of test component

# **Bolted joints with tailored fiber orientation in literature**



#### 2006 Improving the Efficiency of Fiber **Steered Composite Joints using** Load Path Trajectories\* R. LI, D. KELLY, I, A. CROSKY, H. SCHOEN AND L. SMOLLICH School of Mechanical and Manufacturing Engineering University of New South Wales, Sydney 2052, Australia <sup>2</sup>School of Materials Science and Engineering University of New South Wales, Sydney 2052, Australia (Received February 16, 2005) - B90-1 .. B90-2 B90-3 Figure 2. Principal stress trajectories for fiber placemen HTA70-2 - HTA70-4 Displacement (mm) Figure 11. Test curves for 70 mm wide steered fiber specimens.

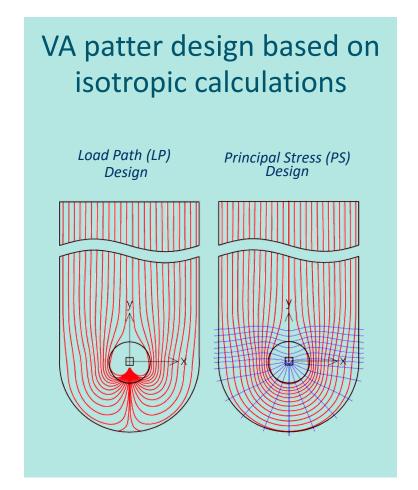




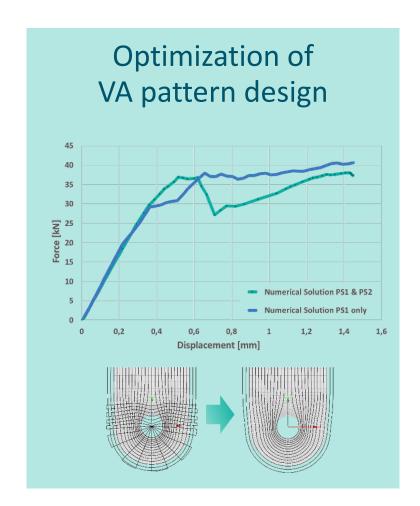
### **Bolted joints with tailored fiber orientation**



Steps for optimizing design process



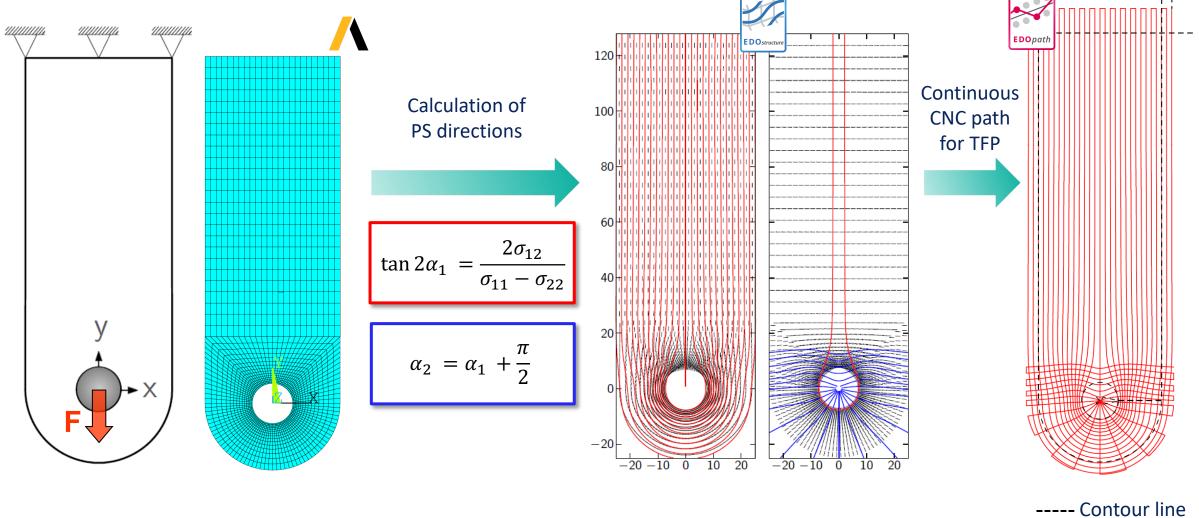




# VA pattern design based on isotropic calculations



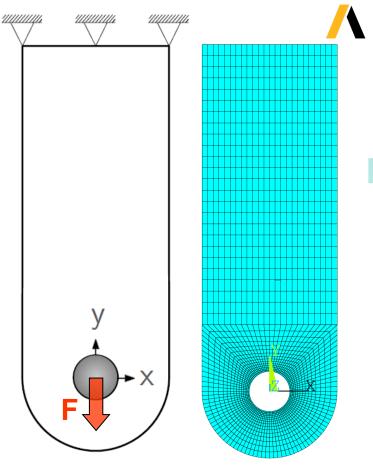
Principal Stress (PS) Design



# VA pattern design based on isotropic calculations

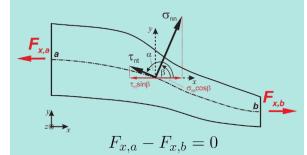


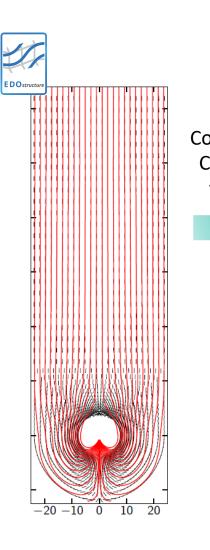
Load Path (LP) Design [1]

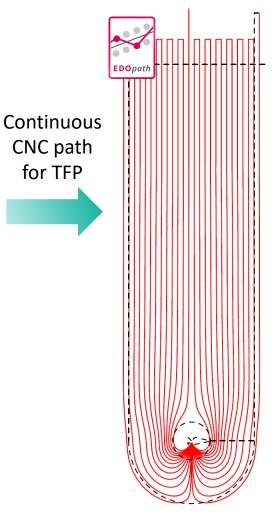


Calculation of Load Path directions

$$\tan \alpha_y = \frac{\sigma_{12}}{\sigma_{11}}$$



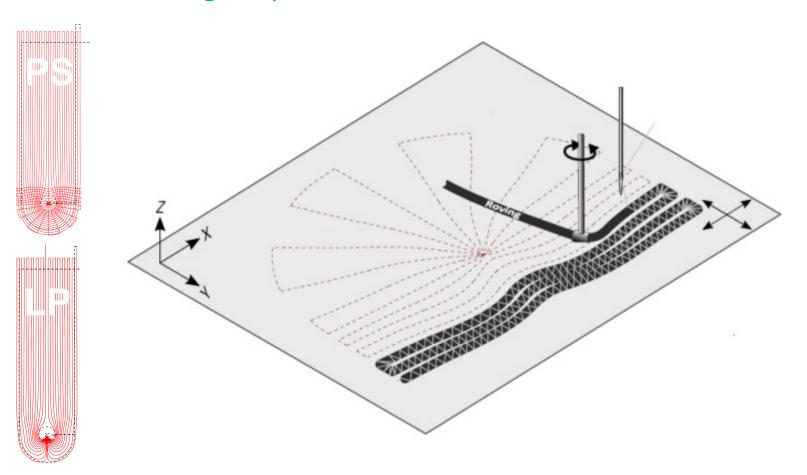




---- Contour line

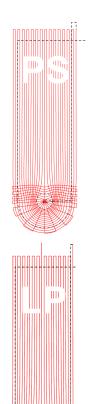


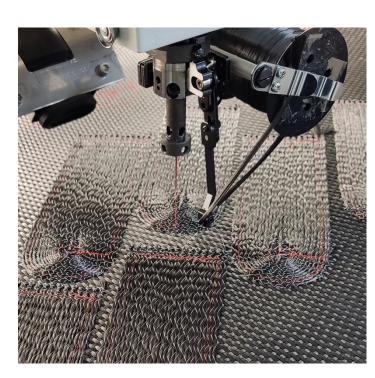
### Manufacturing of specimen





### Manufacturing of specimen





Roving material: CF-HT, 400 tex (6k)

Base material: CF-HT woven fabric,

 $m_A = 380 \text{ g/m}^2$ 

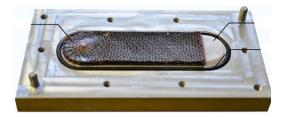


Thickness adapted RTM molds











Epoxy resin: L20 + EPH 161



### Specimen specification and experimental setup [2]



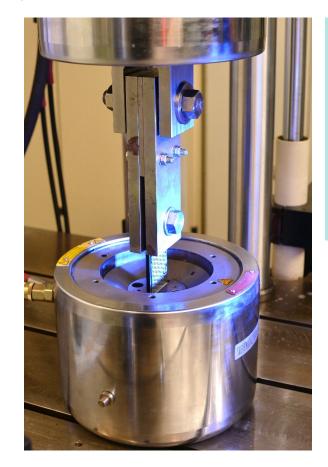
 $[0/90/\pm45]_{4S}$  (base material only)



 $[LP/\pm 45]_{4S}$ 



[PS2/PS1/±45]<sub>4S</sub>



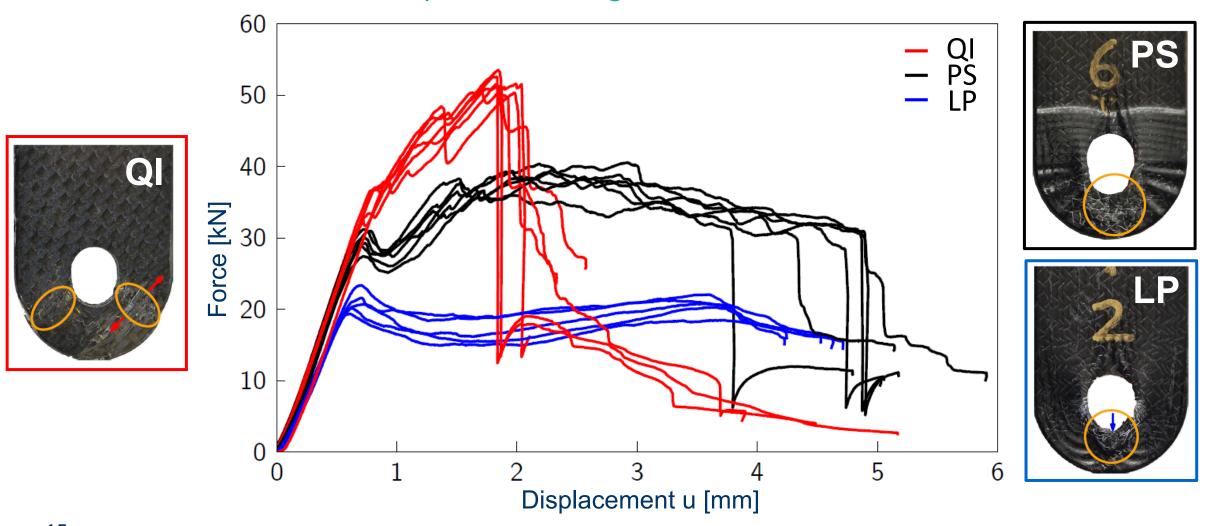
#### **Test setup**

- Instron 8088 with hydraulic clamping
- 250 kN load cell
- Preload: 50 N
- Test speed 1 mm/min
- Number of specimens: 6

Variable-axial specimens are about 22 % lighter than QI type

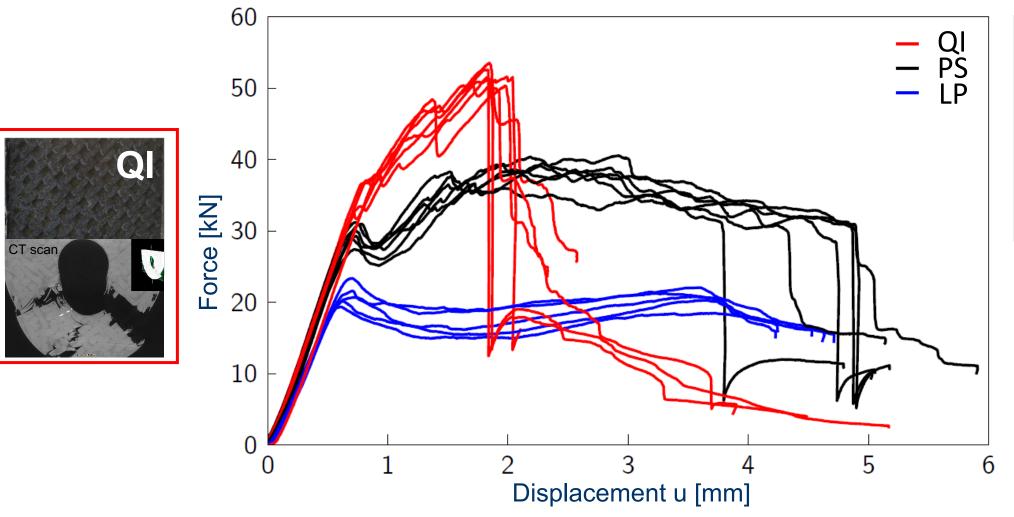


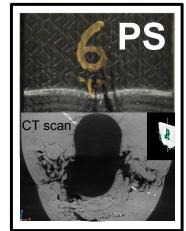
Results: Tensile test force-displacement diagram

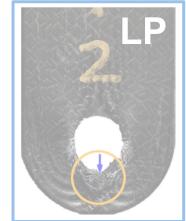




Results: Tensile test force-displacement diagram

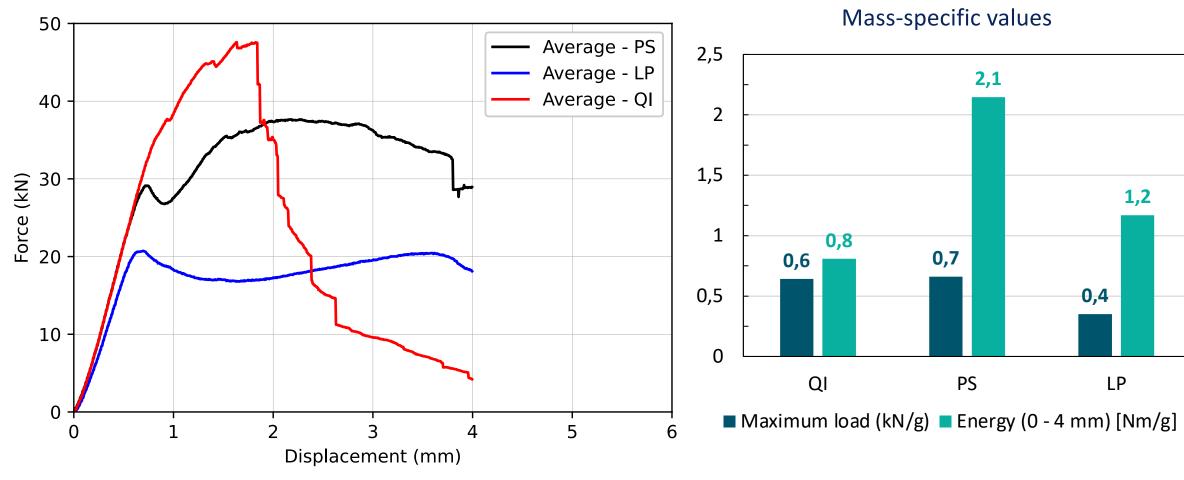






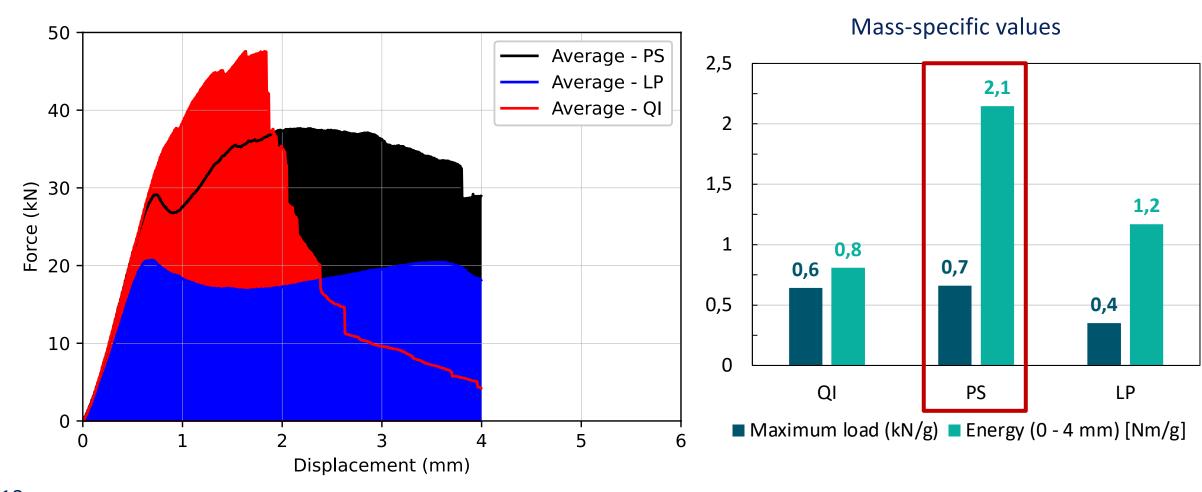


Results: Quasi-plastic deformation energy





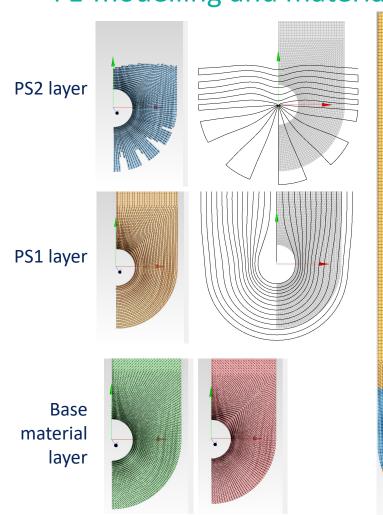
Results: Quasi-plastic deformation energy



### **Model calibration**

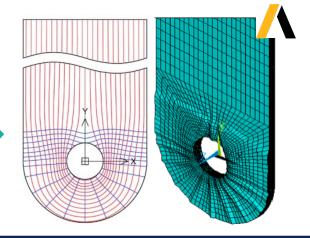


FE-modelling and material parameters



#### Numerical material parameters ( $\phi$ = 55 %)

Fiber, Matrix	Composite
E <sub>  ,f</sub> = 238,000 Mpa	σ <sub>  ,t</sub> = 1,409 Mpa
E <sub>⊥,f</sub> = 16,000 MPa	σ <sub>  ,c</sub> = -7,40 MPa
G <sub>⊥,f</sub> = 50,000 MPa	$\sigma_{\perp,\mathbf{t}}$ = 80 MPa
$v_{\perp \parallel,f}$ = 0.27	$\sigma_{\perp,c}$ = -140 MPa
E <sub>M</sub> = 3,150 MPa	$\tau_{\perp \parallel}$ = $\tau_{\perp \perp}$ = 69 MPa
G <sub>M</sub> = 1,150 MPa	
$v_{M} = 0.37$	



### **Model calibration**

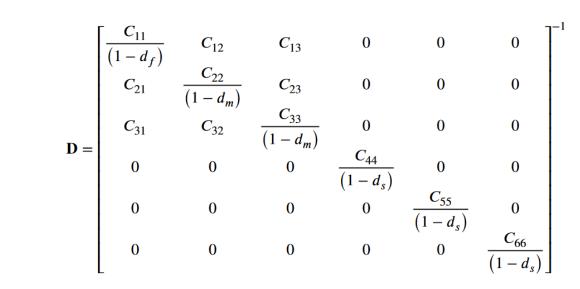


#### Progressive damage simulation with parameter identification by FEMU

#### HASHIN Failure Criterion [3]

$$\begin{aligned} &\text{Fiber} \ \begin{cases} X_{FT} = \left(\frac{\sigma_{11}}{F_{1T}}\right)^2 + \left(\frac{\sigma_{12}^2 + \sigma_{13}^2}{F_6}\right) & \sigma_{11} \geq 0 \\ X_{FC} = \left(\frac{\sigma_{11}}{F_{1C}}\right)^2 & \sigma_{11} < 0 \end{cases} \\ &\begin{cases} X_{MT} = \left(\frac{\sigma_{22}}{F_{2T}}\right)^2 + \left(\frac{\sigma_{23}}{F_4}\right)^2 + \frac{\sigma_{12}^2 + \sigma_{13}^2}{F_6^2} & \sigma_{22} \geq 0 \\ X_{MC} = \frac{\sigma_{22}}{F_{2C}} \left[\left(\frac{F_{2C}}{2F_4}\right)^2 - 1\right] + \left(\frac{\sigma_{22}}{2F_4}\right)^2 \\ &+ \left(\frac{\sigma_{23}}{F_4}\right)^2 + \frac{\sigma_{12}^2}{F_6^2} & \sigma_{22} < 0 \end{cases} \end{aligned}$$

#### Progressive damage modelling [3]



$$d_{f} = \begin{cases} d_{ft} & \sigma_{11} \ge 0 \\ d_{fc} & \sigma_{11} < 0 \end{cases} \qquad d_{m} = \begin{cases} d_{mt} & \sigma_{22} \ge 0 \\ d_{mc} & \sigma_{22} < 0 \end{cases}$$
$$d_{s} = 1 - (1 - d_{ft}) (1 - d_{fc}) (1 - d_{mt}) (1 - d_{mc})$$

#### **Model calibration**

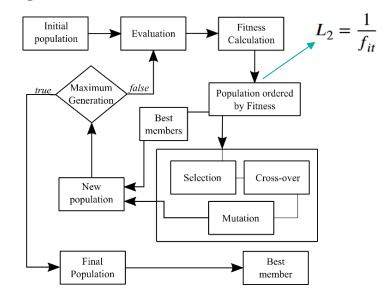


#### Progressive damage simulation with parameter identification by FEMU

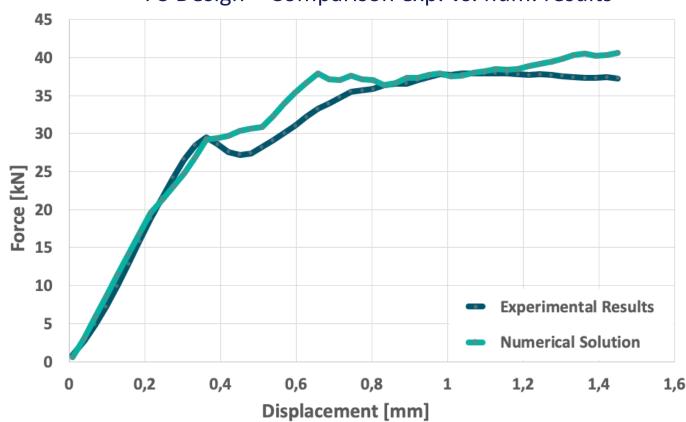
Stiffness degradation parameters [4]  $\rho = \{d_{ft} \quad d_{fc} \quad d_{mt} \quad d_{mc}\}$ 

$$\min_{\rho \in [0,1]} L_2(\rho) = \min_{\rho \in [0,1]} \quad \frac{1}{n_{ptos}} \sqrt{\sum_{i=1}^{n_{ptos}} \left(\frac{y_i^E - y_i^N(\rho)}{y_i^E}\right)^2}$$

#### Genetic Algorithm-based [4]



#### PS Design – Comparison exp. vs. num. results



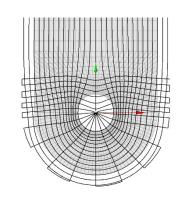
#### **Conclusion and Outlook**

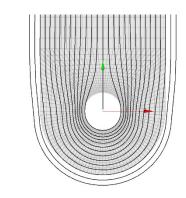
Bolt joint loaded composites with a variable-axial fiber design shows improved quasi-plastic behavior compared to QI multi-axial laminate.

- Optimality based PS design shows higher mass energy absorption than LP design
- By help of experimental tests a progressive damage modelling setup was identified

Optimization of variable-axial fiber patterns by using, e.g., DFPO [5]

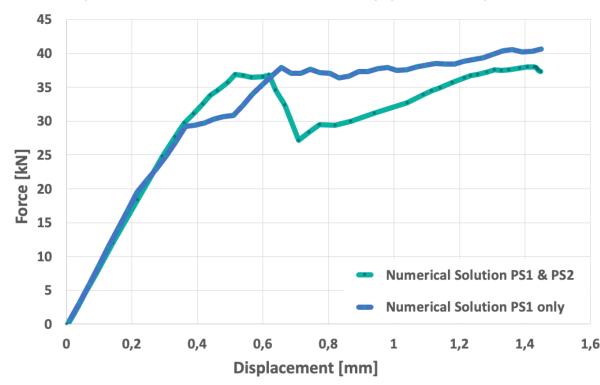
- Optimizing for maximal mass-specific loads
- Optimizing for even higher energy absorption







Comparison PS1+PS2 vs. PS1 only pattern specimen





# ipf

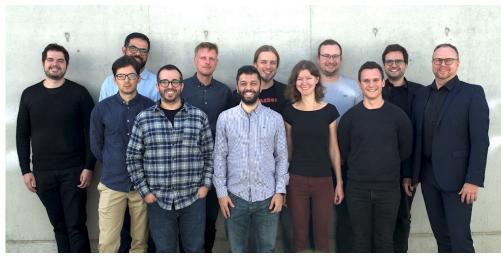
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# Thank you for your attention