



# Gap Formation and Resin Flow in Bent Preforms for Resin Transfer Moulding

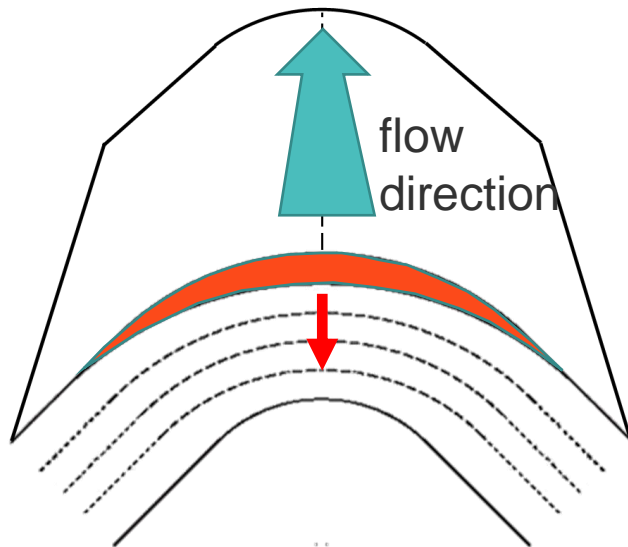
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## Manufacture of composite components employing Resin Transfer Moulding

Localised reinforcement compression at bends in component

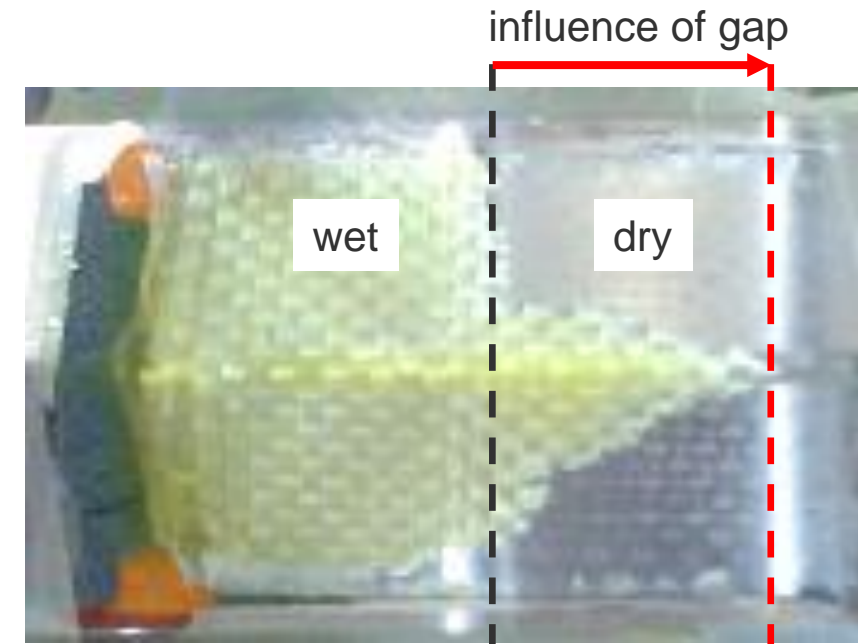
➡ formation of gaps between reinforcement and tool surface



Experiment: Perspex tool, 90° bend

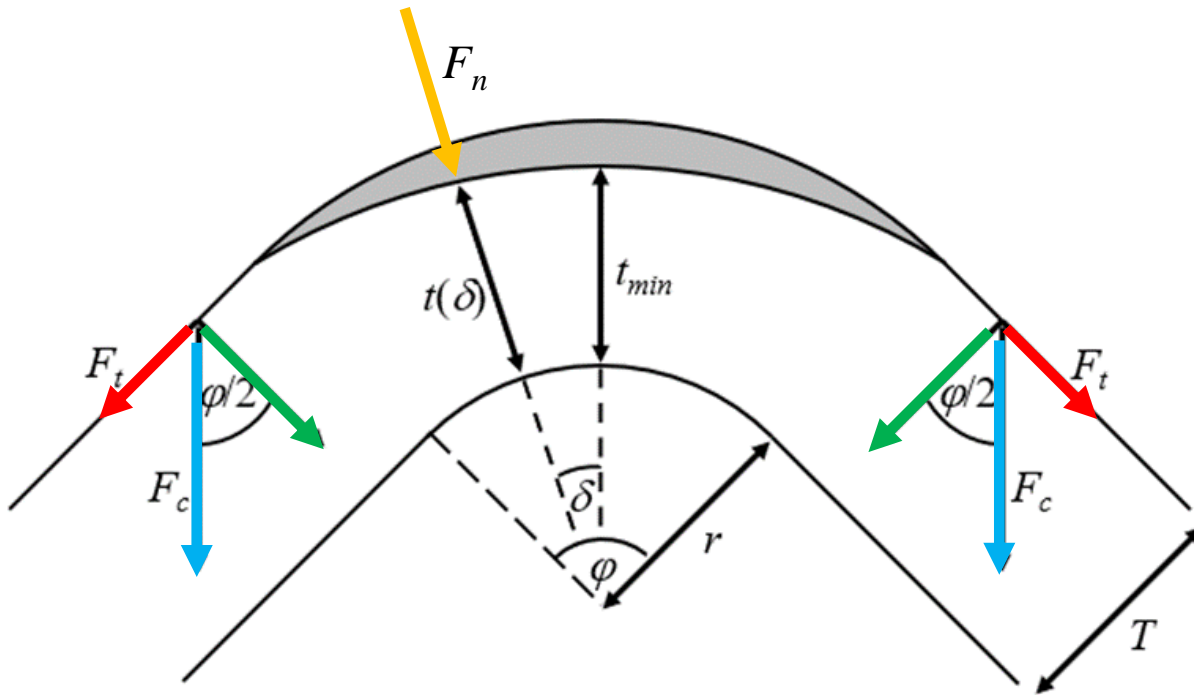
Observation:

Local effect on flow front propagation; racetracking makes process hard to control



Koutsonas, Spiridon (2015) Race-track modelling and variability in RTM for advanced composites structures. PhD thesis, University of Nottingham.

# Gap formation



geometry parameters:  $\varphi, r, T$  (and  $L$ )

material parameters:  $\mu, a, b$

tool closing force:  $\underline{2F_c}$

tool closing force:  $\underline{F_c} = \frac{F_0(T, a, b)}{\cos(\varphi / 2)}$

tensile force:  $\underline{F_t} = \mu \underline{F_c} \cos(\varphi / 2)$

normal force:  $\underline{F_n(\delta)}$  **model?**

local pressure:  $p = \frac{d \underline{F_n}}{d A}$

reinforcement compressibility:  $t(\delta) = a \left( \frac{p(\delta)}{10^5 \text{ Pa}} \right)^{-b}$

Describe gap in terms of

- Minimum reinforcement thickness (at  $\delta = 0$ ),  $t_{min}$
- Opening half angle (at  $t = T$ ),  $\delta_T$

# Gap formation (experiments)

Three points on reinforcement surface are known:

$$\delta = \pm \delta_T, \quad t = T \quad \text{and} \quad \delta = 0, \quad t = t_{min}$$

Describe surface as circle,

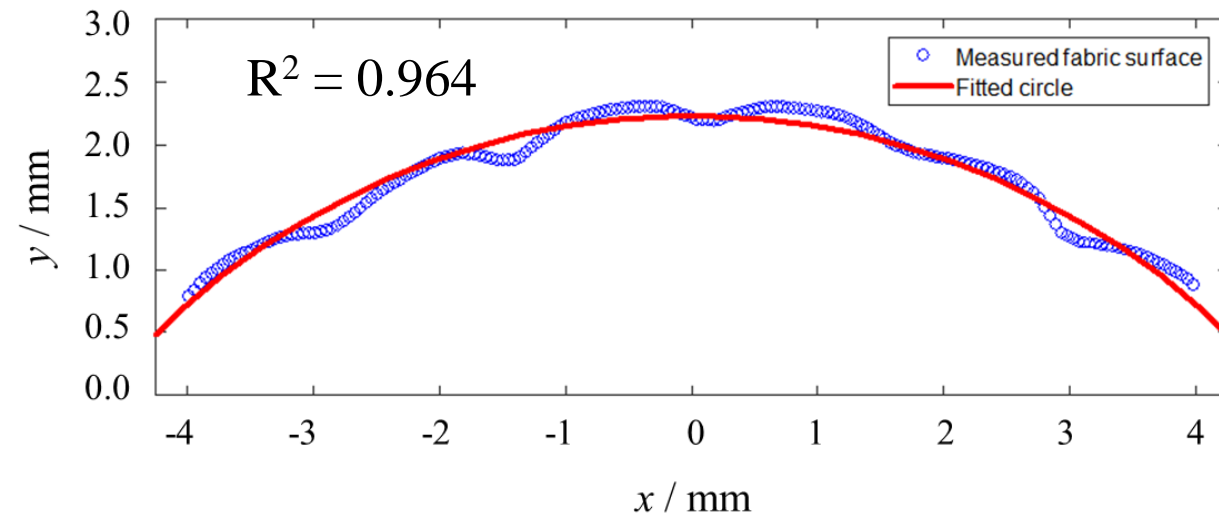
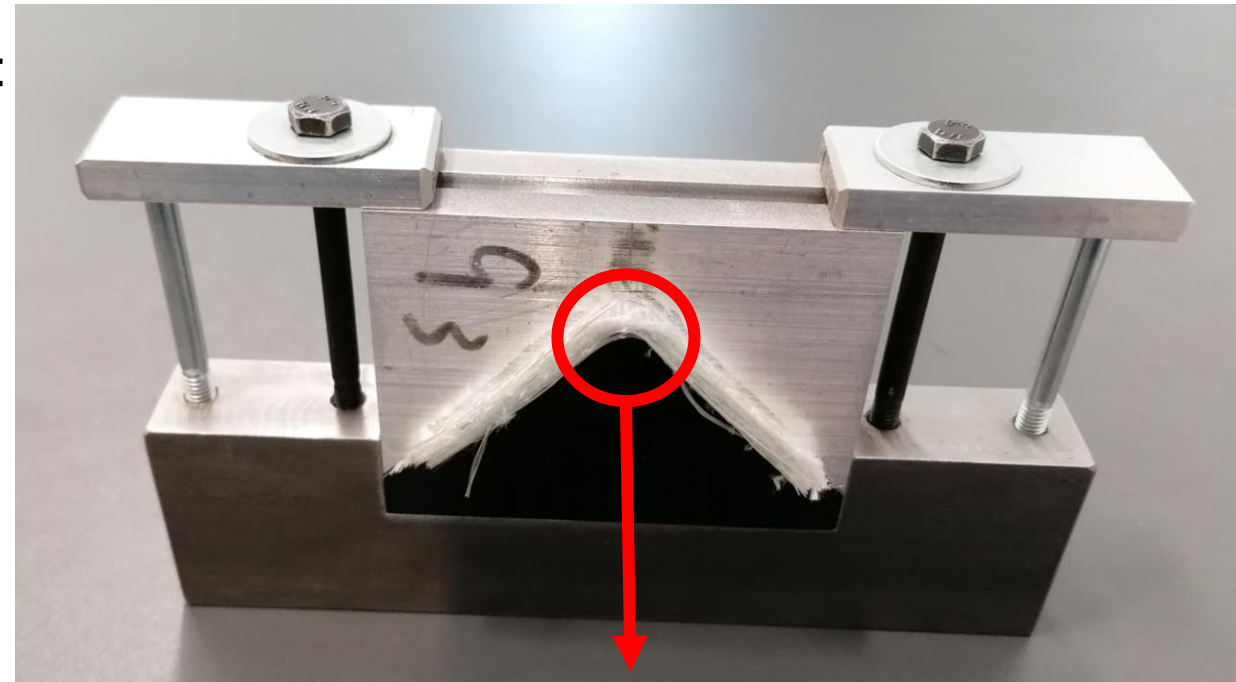
- with radius

$$R = \frac{2(r+T)(r+t_{min})\cos\delta_T - (r+T)^2 - (r+t_{min})^2}{2((r+T)\cos\delta_T - (r+t_{min}))}$$

- and offset (of centre point)

$$x_c = \frac{(r+T)^2 - (r+t_{min})^2}{2((r+T)\cos\delta_T - (r+t_{min}))}$$

Experimental data suggest that this is a reasonable description



# Effective permeability (analytical)



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For axial flow in gap, solve Poisson equation

$$C = \frac{1}{\rho} \frac{\partial}{\partial \rho} \left( \rho \frac{\partial u}{\partial \rho} \right) + \frac{1}{\rho^2} \frac{\partial^2 u}{\partial \theta^2}$$

$$C = \frac{1}{\eta} \frac{\Delta p}{L}$$

**Special case:** “moon shaped” duct

$$x_c = -(r+T) \quad \text{and} \quad R = 2r+T+t_{min}$$

$$\delta_T = \arccos \frac{(r+t_{min})^2 - (r+T)^2 + 2(r+T)(r+t_{min})}{2(r+T)^2}$$

Analogy to torsion of bars (Timoshenko)

$$u(\rho, \theta)$$

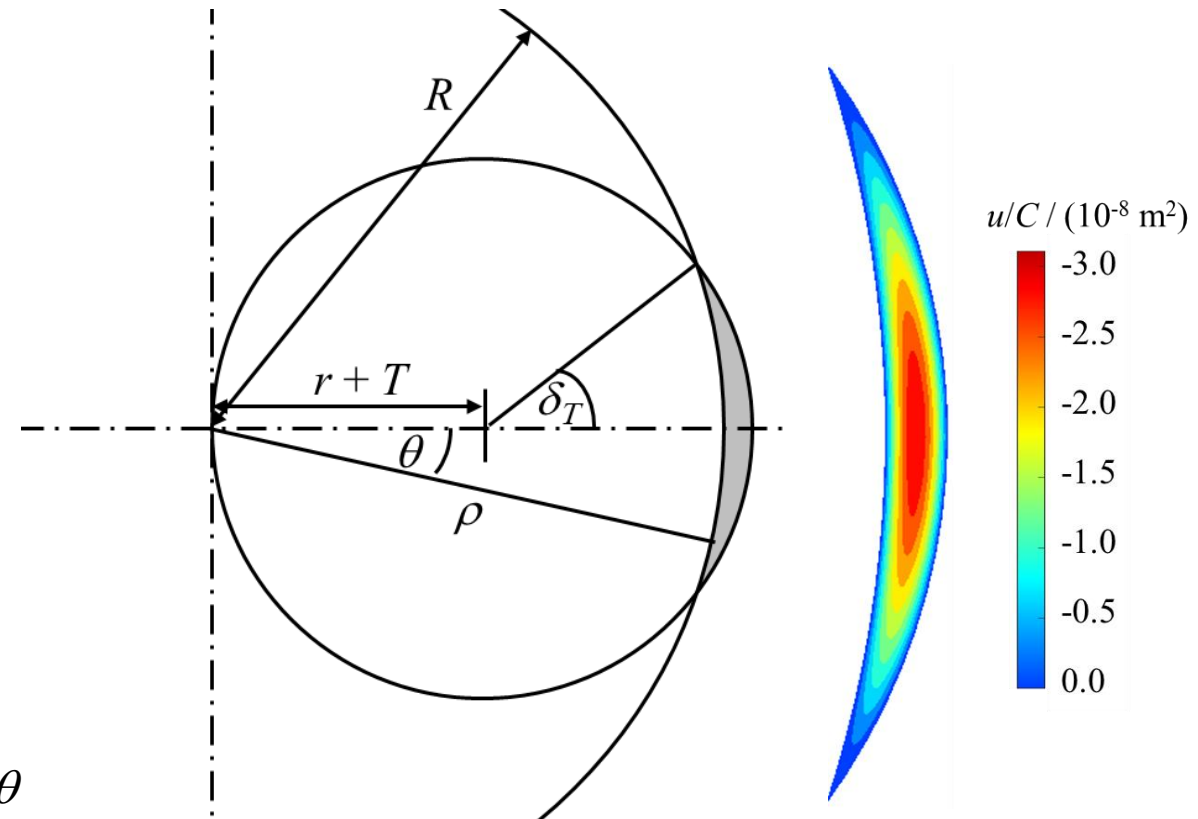
$$= C \frac{1}{4} \rho^2 - C \frac{r+T}{2} \rho \cos \theta + C \frac{(r+T)R^2}{2} \frac{1}{\rho} \cos \theta - C \frac{R^2}{4}$$

Calculate

$$Q = \int_{-\theta_T}^{\theta_T} \int_R^{2(r+T)\cos\theta} u \rho d\rho d\theta \quad \text{and} \quad A = \int_{-\theta_T}^{\theta_T} \int_R^{2(r+T)\cos\theta} \rho d\rho d\theta$$



$$K_g = -\frac{Q}{CA}$$

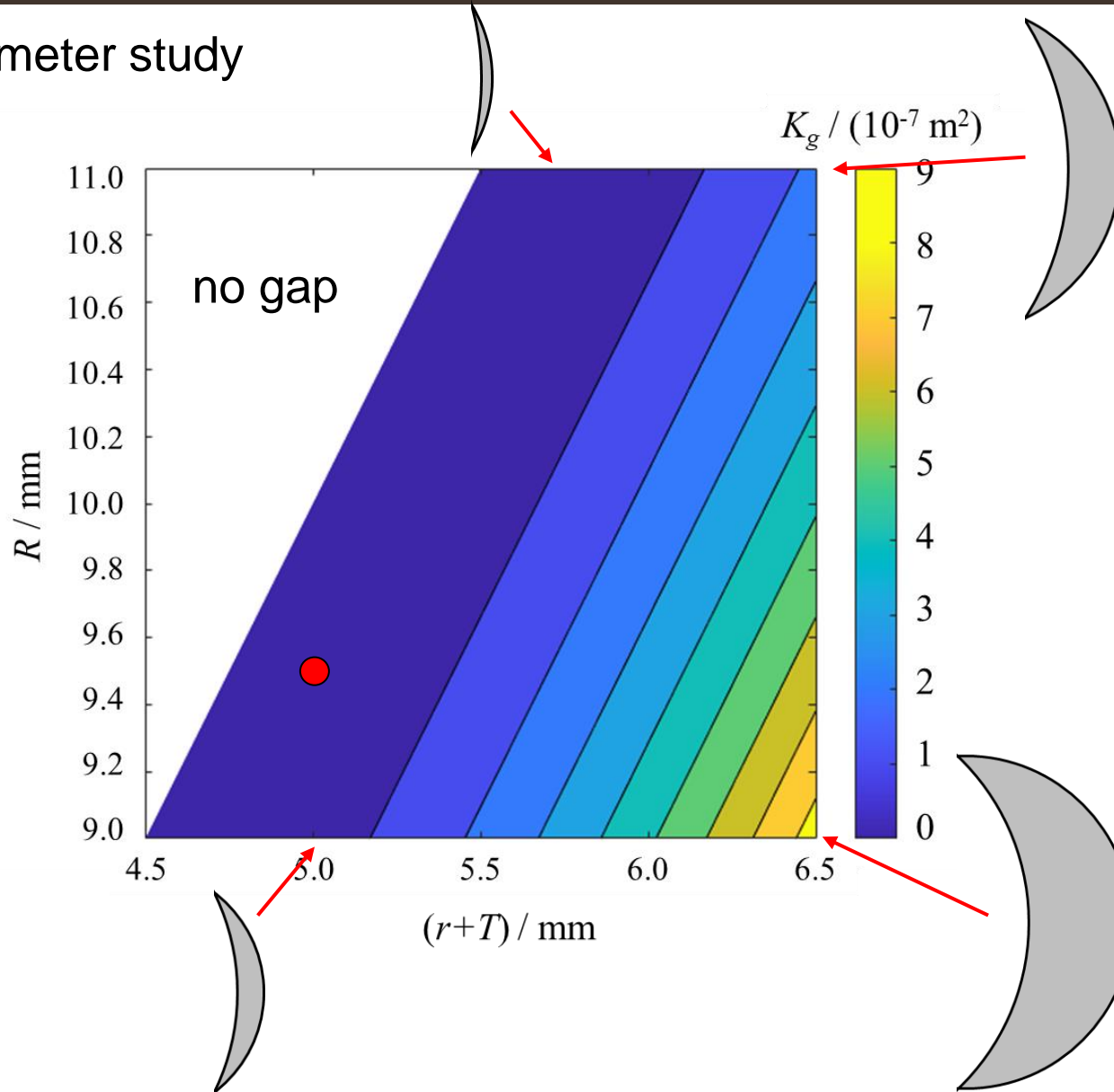


$$r+T = 5.0 \text{ mm} , R = 9.5 \text{ mm}$$

$$\text{Gap only: } K_g = 1.41 \times 10^{-8} \text{ m}^2$$

# Effective permeability (analytical)

## Parameter study



## From Darcy-Weisbach

$$K_g = \frac{2 D_h^2}{c}$$

## Hydraulic diameter

$$D_h = \frac{4 A_g}{P_g}$$

← gap cross-sectional area  
← perimeter

## Shape factor, $c$

- is a measure for the effect of viscous friction on flux;
- is related to the length of boundaries between fluid layers moving at different velocity in laminar flow.

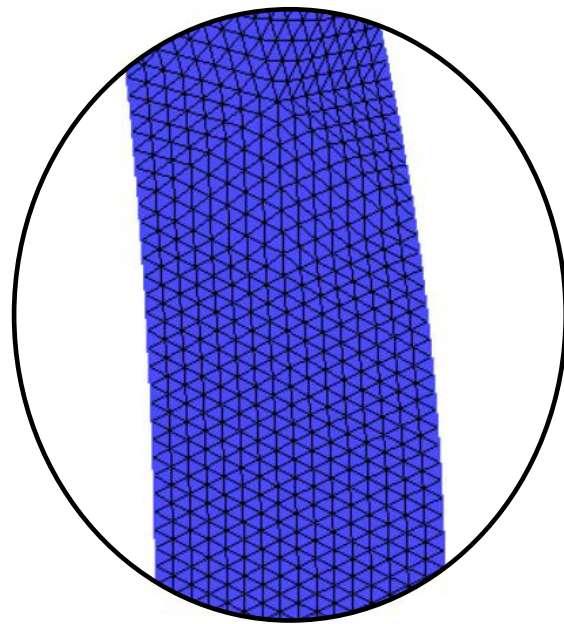


# Effective permeability (numerical)

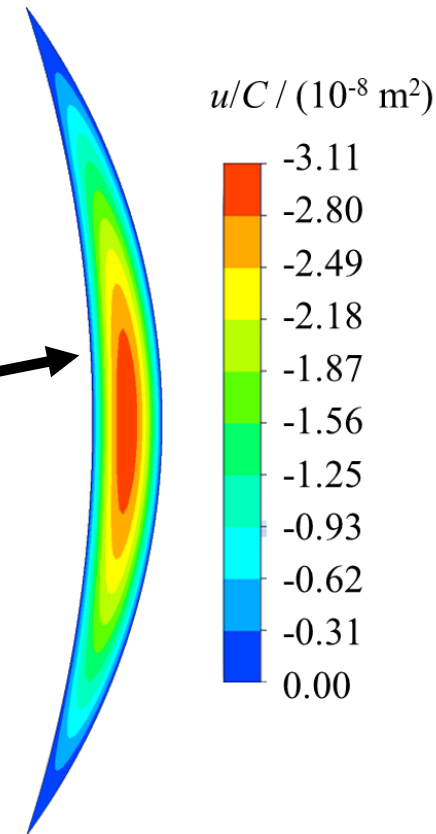
## Validation of steady-state CFD simulation (axial flow in moon shaped gap)

Flow velocity distribution

Fine  
discretisation  
required

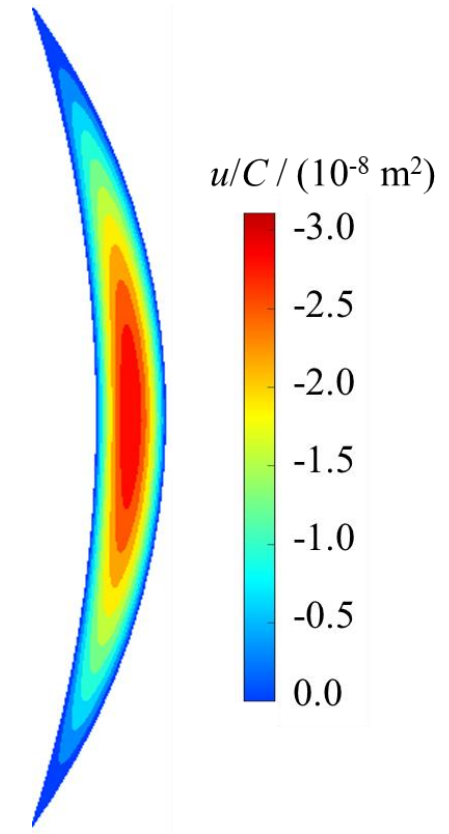


Numerical (Ansys CFX)



➡  $K_g = 1.44 \times 10^{-8} \text{ m}^2$

Analytical



➡  $K_g = 1.41 \times 10^{-8} \text{ m}^2$

Effective permeability

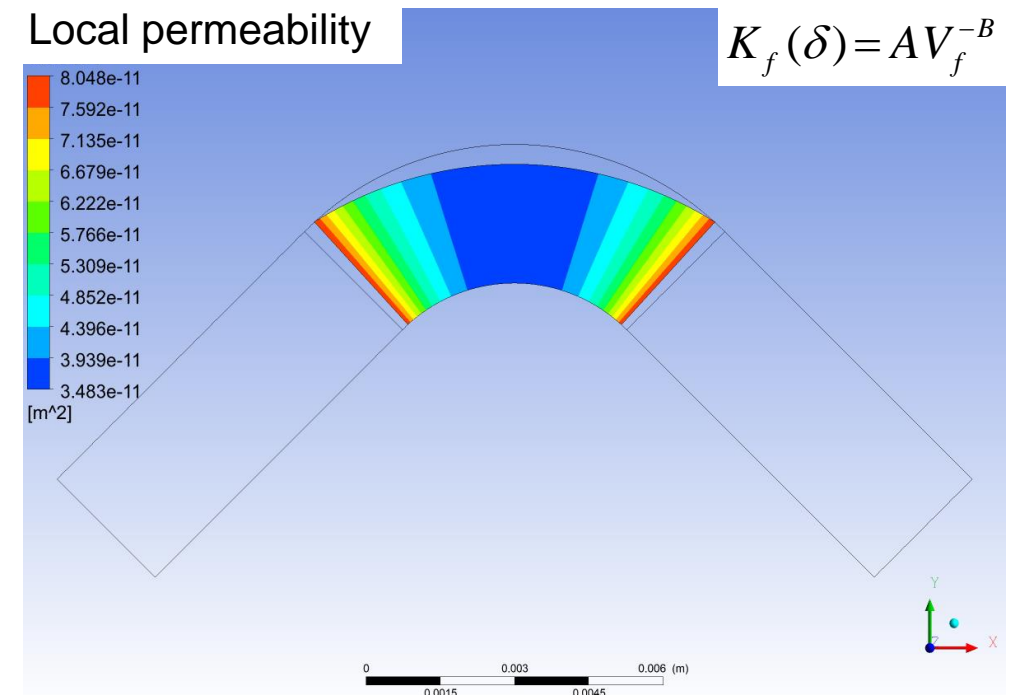
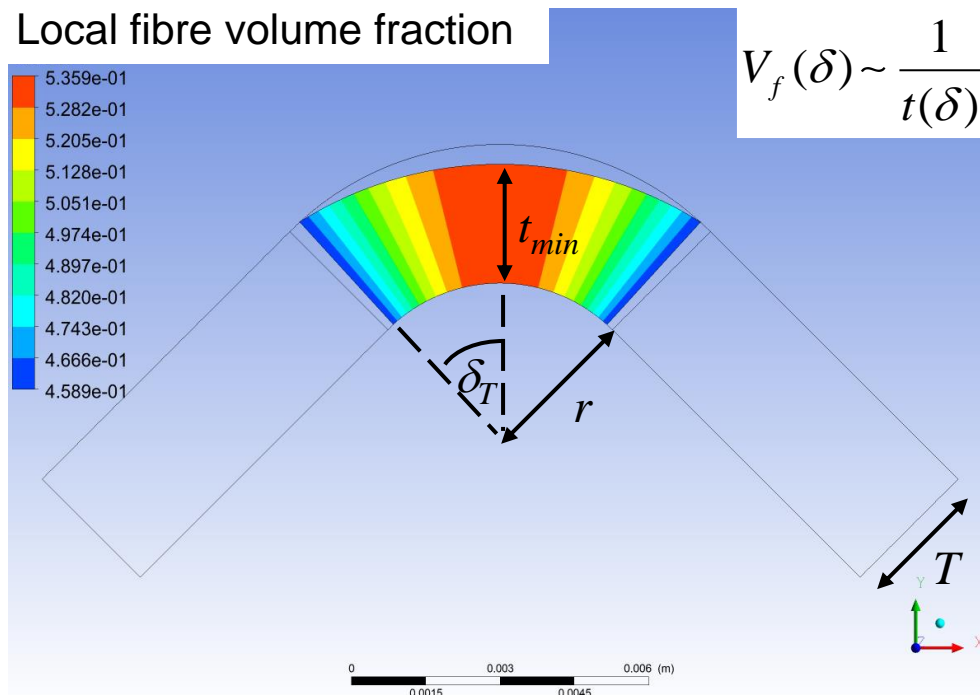
# Effective permeability (simulation)

## Analyse axial and transverse flow through bend using CFD

Reinforcement on both sides of bend: porous medium with uniform properties (thickness  $T$ )

Compressed reinforcement ( $-\delta_T < \delta < \delta_T$ ): porous medium with varying properties (thickness  $t(\delta)$ )

In gap: fluid only





# Effective permeability (simulation)

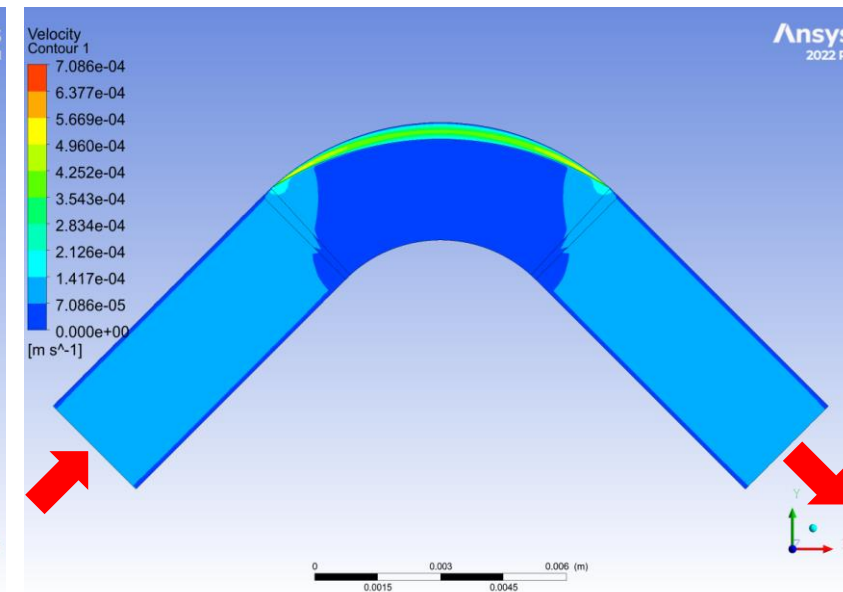
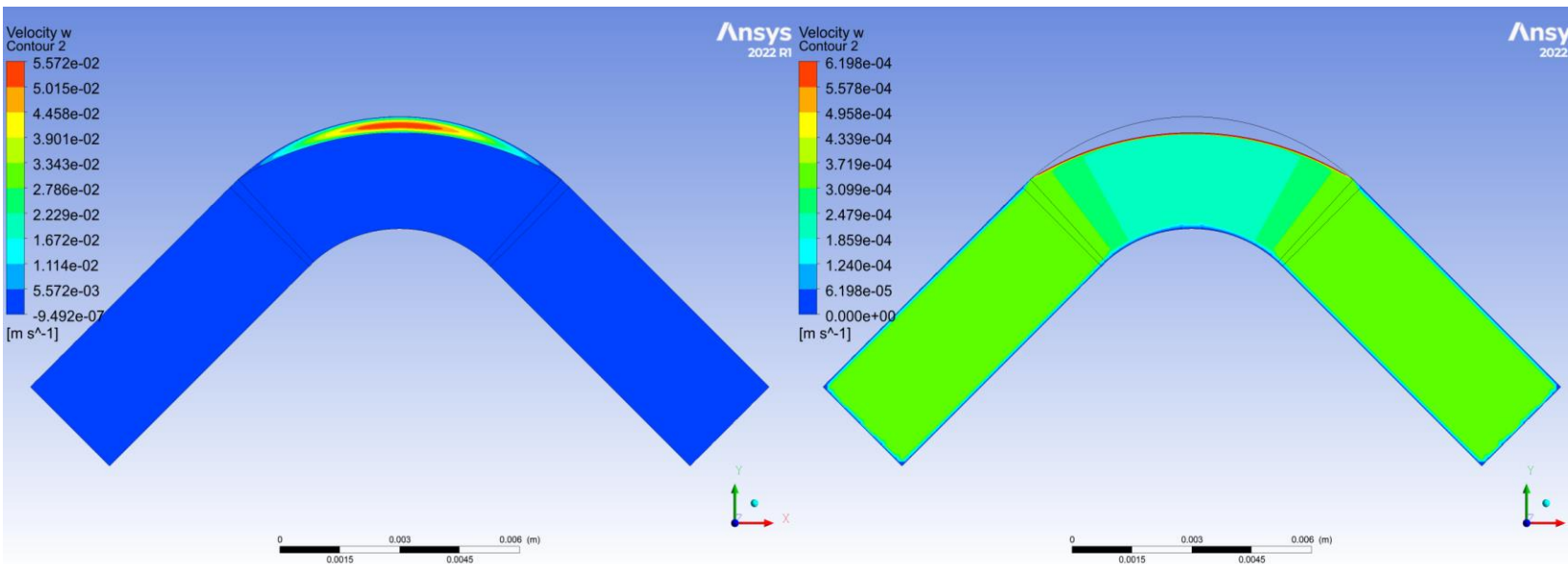
## Steady-state flow simulation (saturated flow)

$\varphi$	$r$ / mm	$T$ / mm	$t_{min}$ / mm	$\delta_T$	$R$ / mm	$x_c$ / mm
90°	3.20	2.80	2.40	42°	7.60	-2.00

Determine permeability from mass flow

axial  $K_a = 5.03 \times 10^{-10} \text{ m}^2$

transverse  $K_t = 9.37 \times 10^{-11} \text{ m}^2$

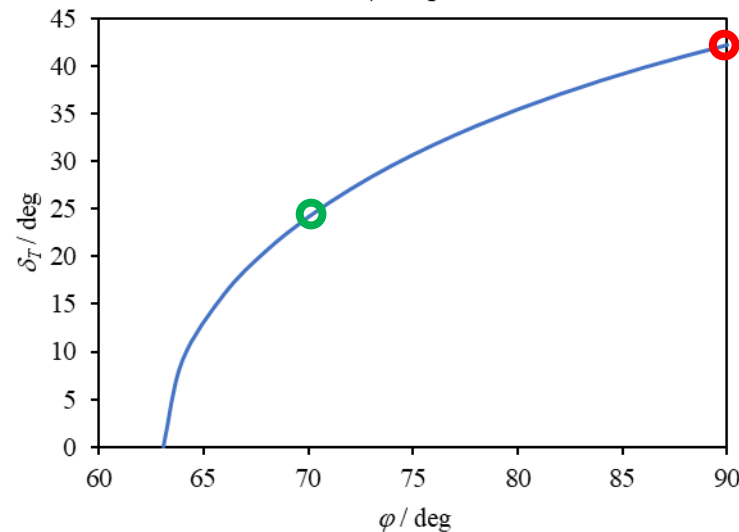
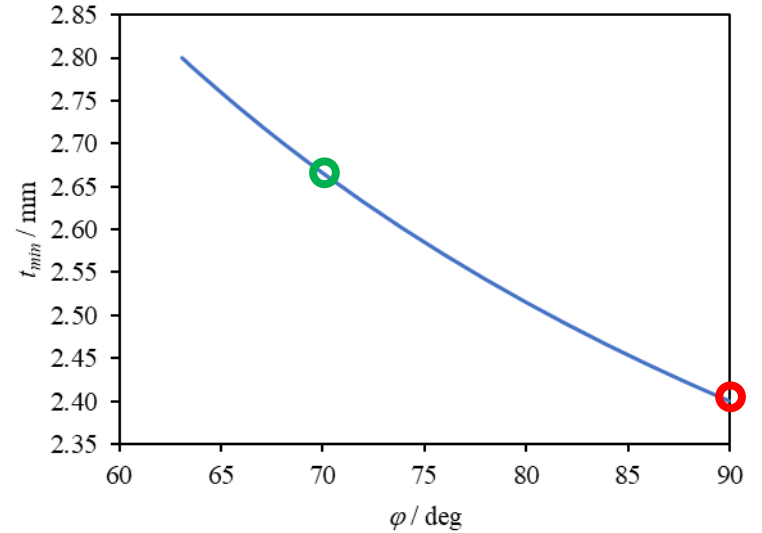


Permeability of flat reinforcement at thickness  $T$  is  $8.01 \times 10^{-11} \text{ m}^2$

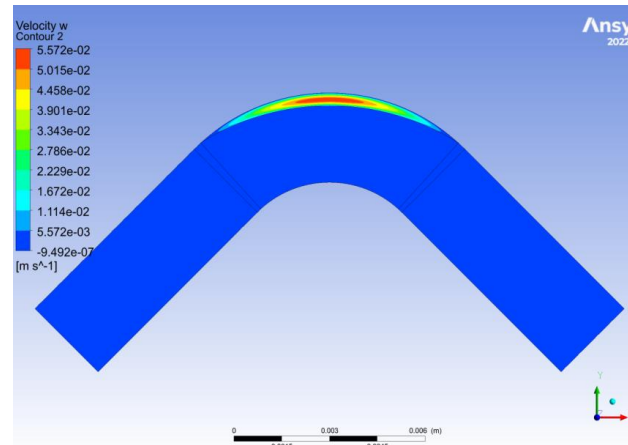
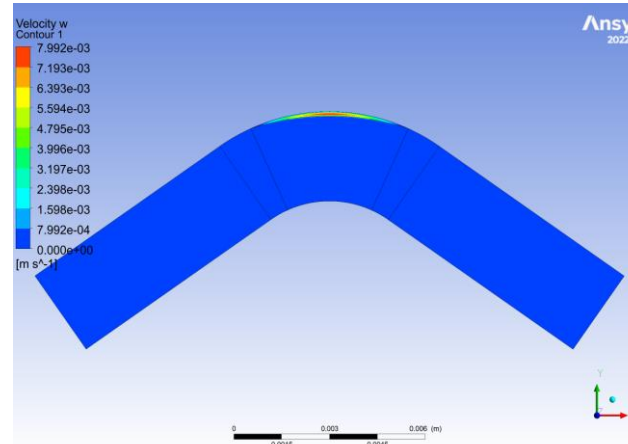
# Effective permeability (simulation)

## Effect of bend angle

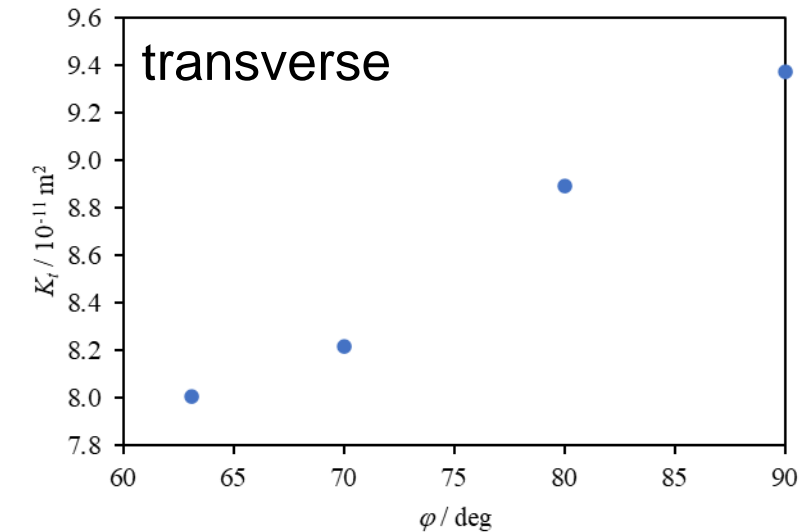
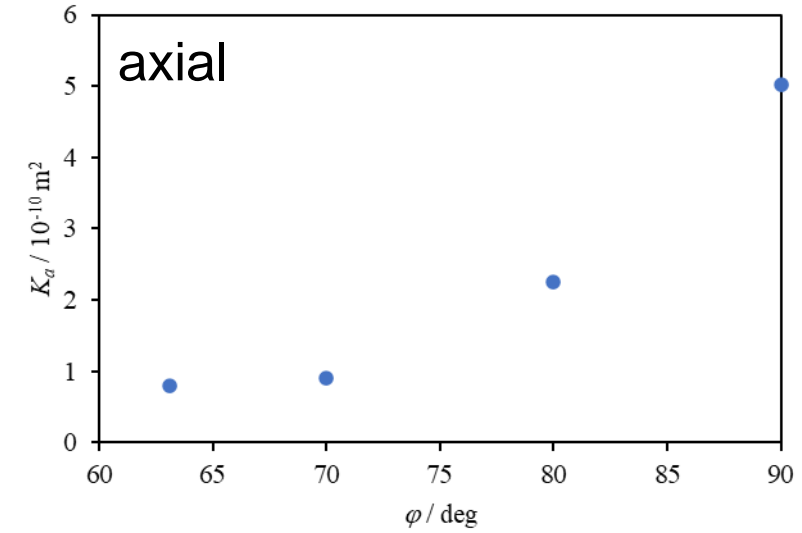
(preliminary) estimate for gap shape



simulation models

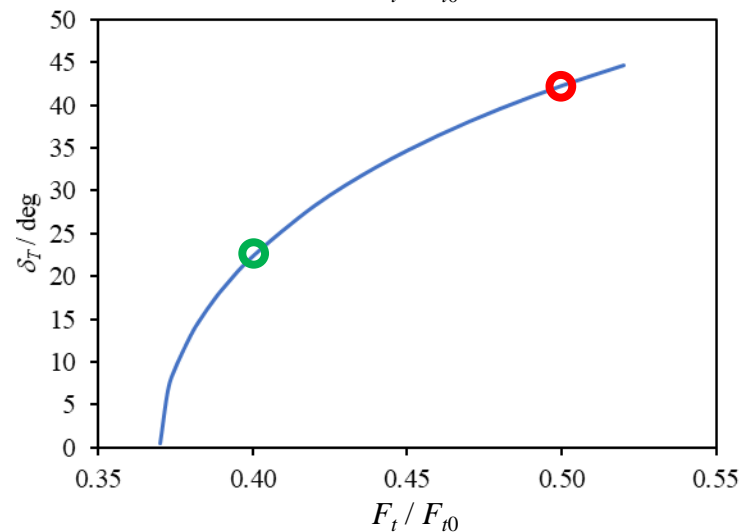
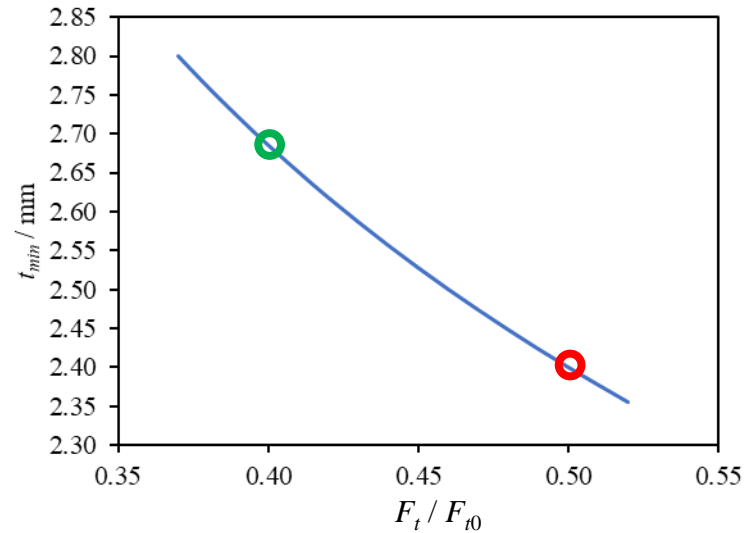


effective permeability

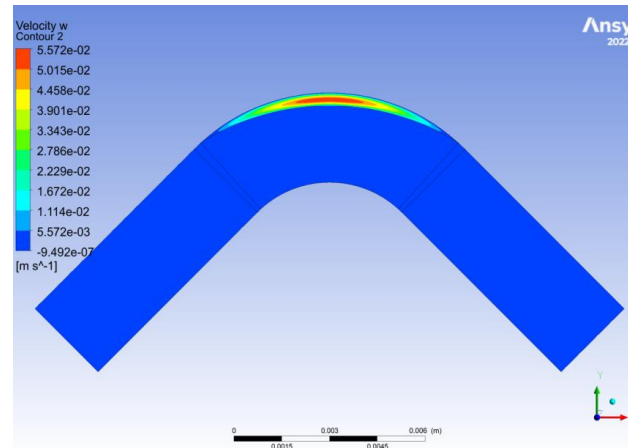
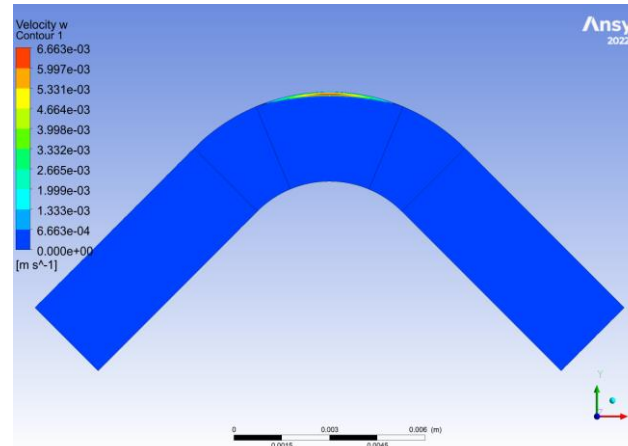


# Effective permeability (simulation)

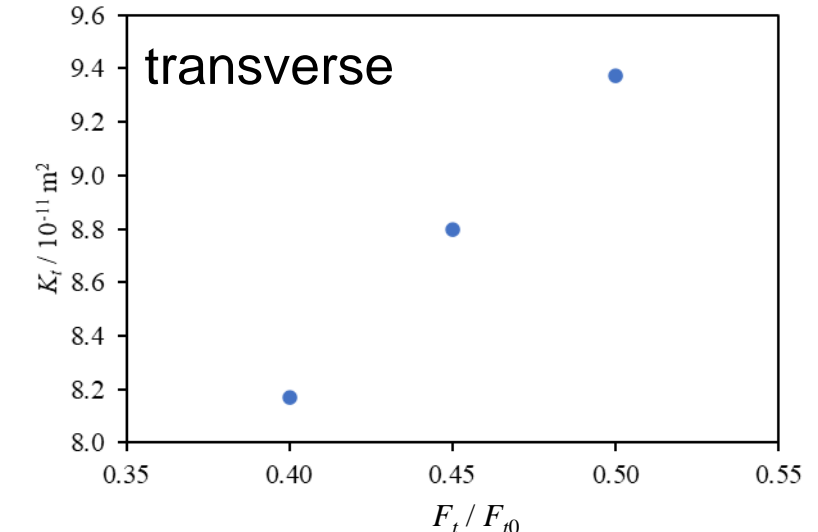
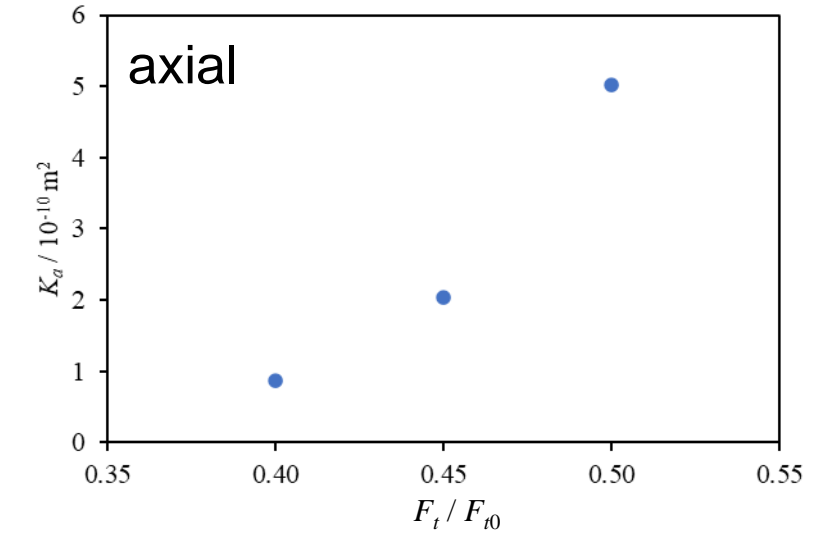
**Effect of (normalised) tensile force**  
(preliminary) estimate for gap shape



simulation models



effective permeability



# Flow front shapes

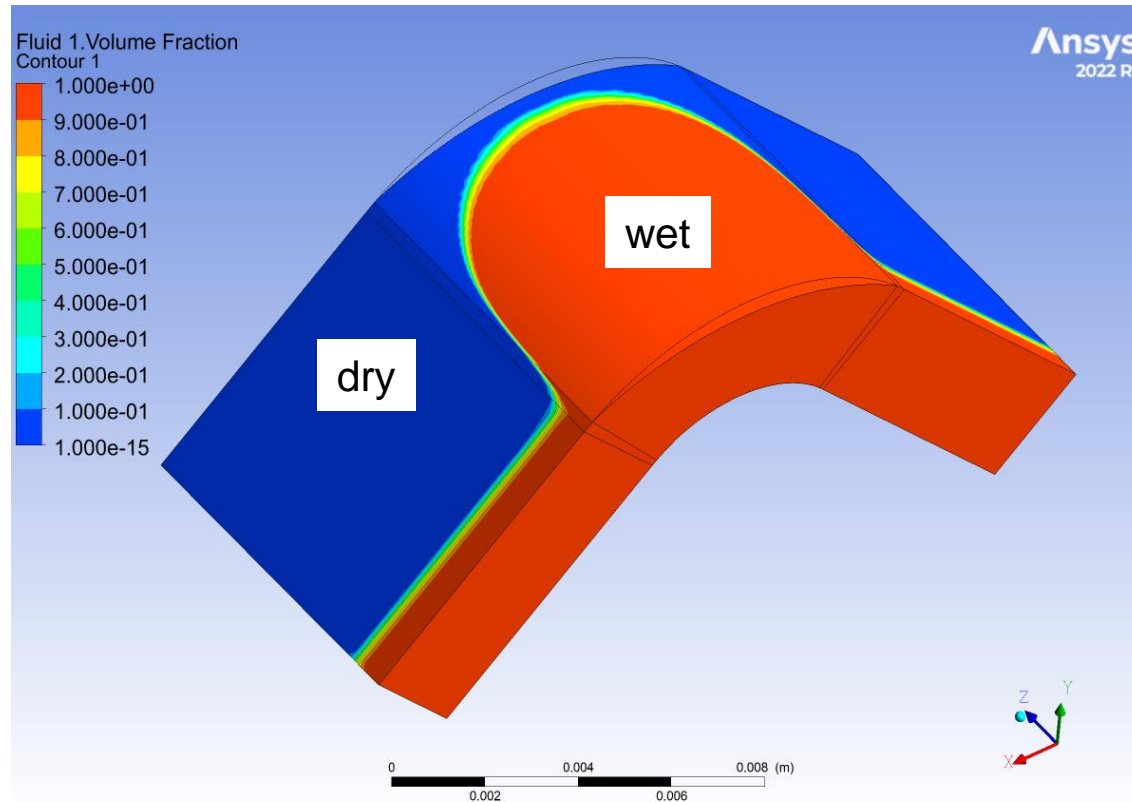


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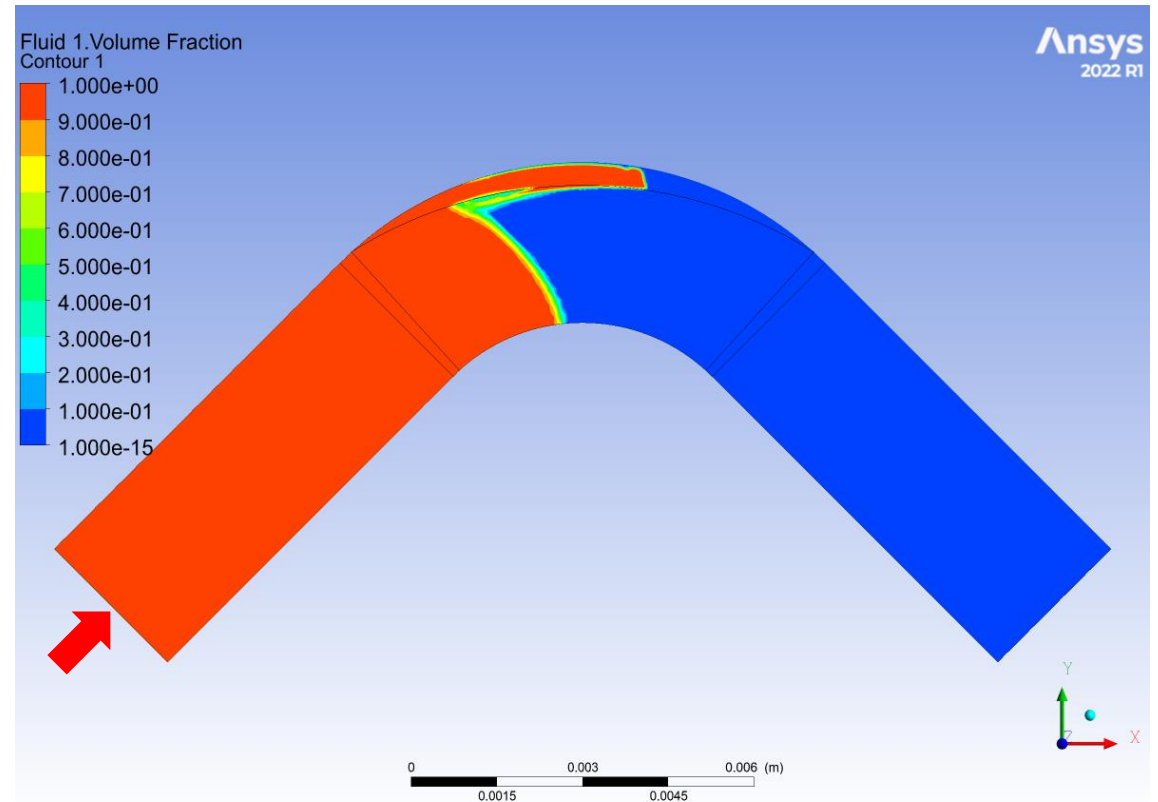
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## Transient flow simulation (unsaturated flow)

axial



transverse



Inside of bend: lagging of flow front (locally reduced reinforcement permeability)

Outside of bend: racetracking in gap

- A model for formation of gap between reinforcement and upper tool surface (and the shape of the gap) still needs to be formulated.
- Descriptors for the gap size were defined, i.e. gap height and opening angle.
- An analytical solution for the effective gap permeability was derived for a special case.
- The effect of different parameters on the gap size was estimated.
- CFD simulations were run to find effective permeabilities of the bend (gap and compressed reinforcement).
- Typical flow patterns at the bend were predicted.

# Acknowledgement

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